



2017智能工业设计技术研讨会  
2017. 10. 26-28  
宁波, 中国

# 超临界CO<sub>2</sub>膨胀机涡轮气动设计优化

诸葛伟林

清华大学

2017年10月27日

# 清华大学热流体研究团队简介

## • 张扬军 教授、博士生导师，长江学者特聘教授



- ◆ 汽车安全与节能国家重点实验室常务副主任、清华大学通用航空技术研究中心主任
- ◆ 军委科技委xx专家组专家，军委装备发展部xx专业组专家
- ◆ 国际杂志I. J. of Fluid Machinery and Systems 主编
- ◆ 获国家科技进步二等奖2项（排名1, 3），国防科技进步一等奖等省部级奖9项

## • 诸葛伟林 副研究员、博士生导师



- ◆ ASME 国际燃气轮机学会 ORC 动力系统委员会委员
- ◆ 中国兵工学会发动机委员会委员，中国内燃机工业协会增压器分会技术委员会委员
- ◆ 《内燃机学报》编委、《船舶工程》编委
- ◆ 获国家科技进步二等奖1项（排名5），国防科技进步一等奖等省部级奖5项

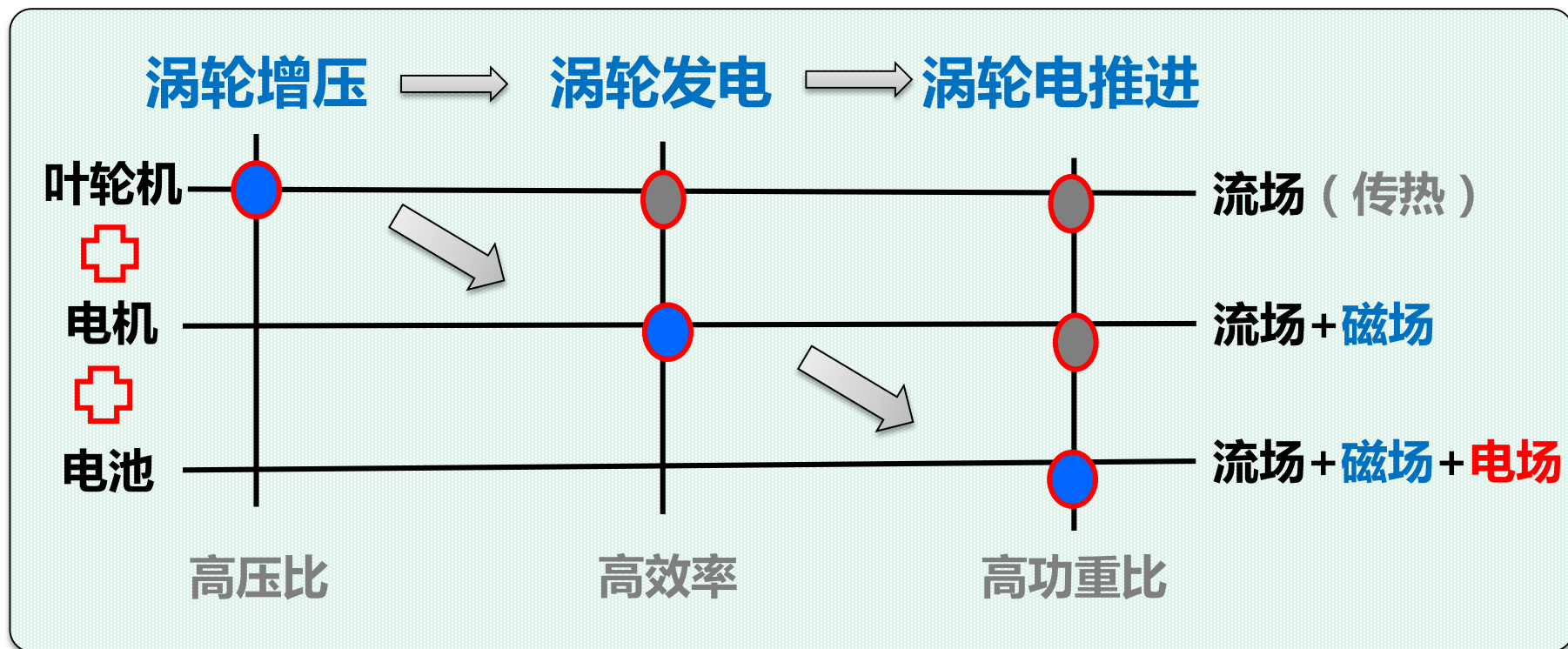
## • 钱煜平 助理研究员、硕士生导师



- ◆ 主持国防973项目专题研究3项
- ◆ 牵头研制的国内首架电动涵道风扇无人机参加“第二届军民融合发展高技术成果展”
- ◆ 清华-亚琛协同创新“航空混合电推进项目”中方技术负责人

# 研究方向与关键技术

- **研究方向**：发动机热流体工程（热力学、流体力学与控制工程交叉融合）
- **关键技术**：涡轮增压、涡轮发电与涡轮电推进



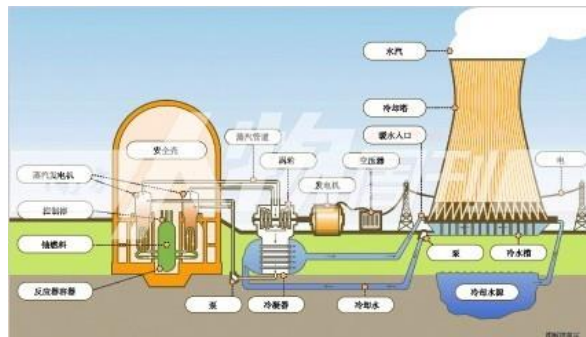
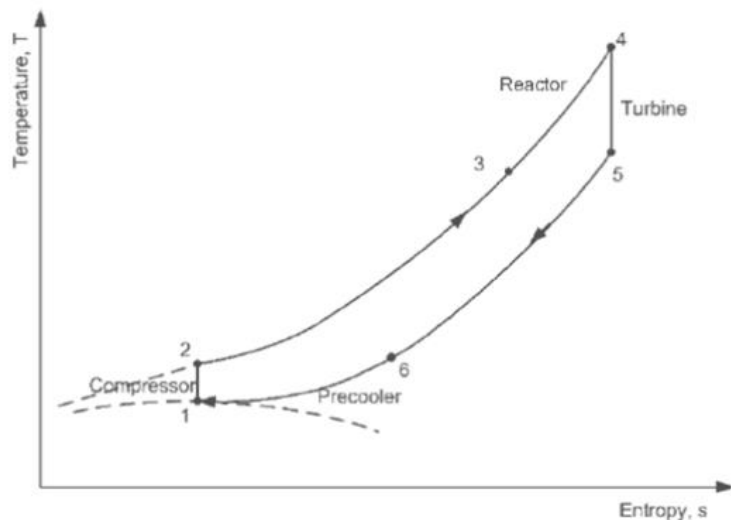
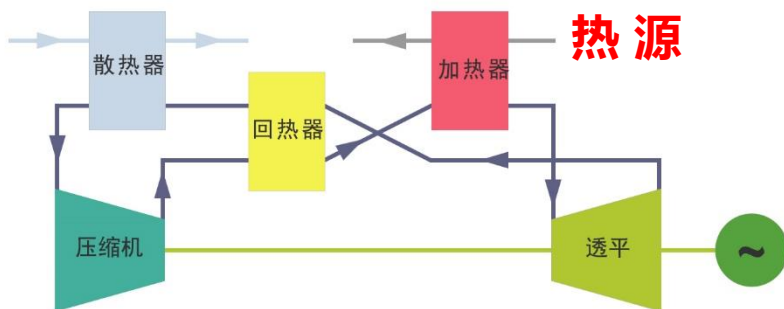
# 目 录

---

- 超临界CO<sub>2</sub>动力循环
- 膨胀机涡轮气动设计
- 膨胀机进气蜗壳优化
- 膨胀机涡轮叶型优化

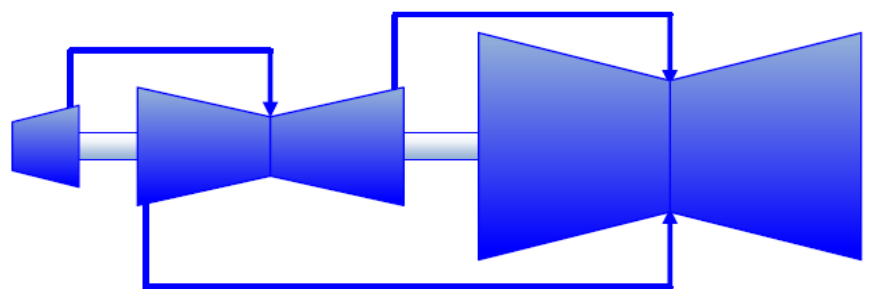
# 超临界CO<sub>2</sub>动力循环

## 超临界CO<sub>2</sub>动力循环原理



## ■ 超临界CO<sub>2</sub>动力循环特点

- 功率密度高
- 高温特性好
- 循环效率高



5 m

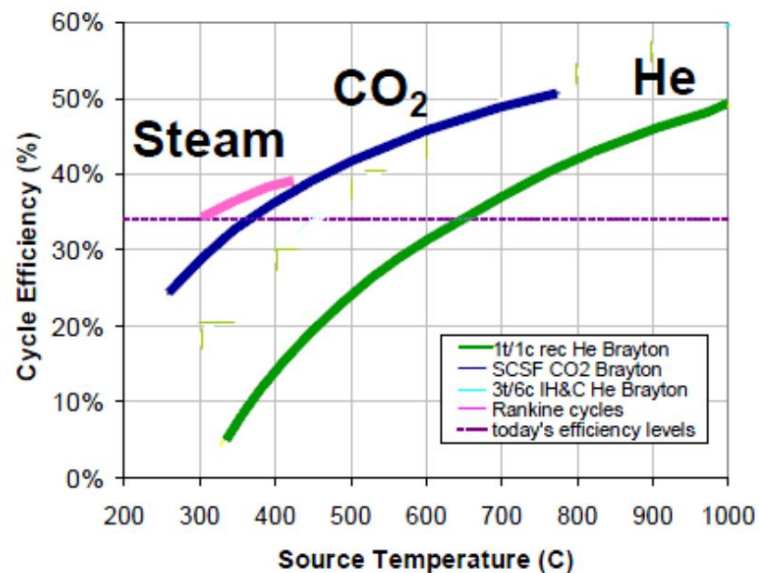
Steam turbine: 55 stages / 250 MW  
Mitsubishi Heavy Industries (with casing)



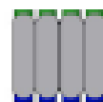
Helium turbine: 17 stages / 333 MW (167 MW<sub>e</sub>)  
X.L. Yan, L.M. Lidsky (MIT) (without casing)

1 m

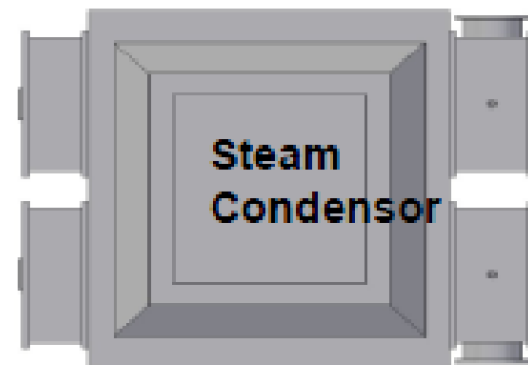
sCO<sub>2</sub> turbine: 4 stages / 450 MW (300 MW<sub>e</sub>)  
(without casing)



s-CO<sub>2</sub>  
Cooler



1 m



Steam  
Condenser

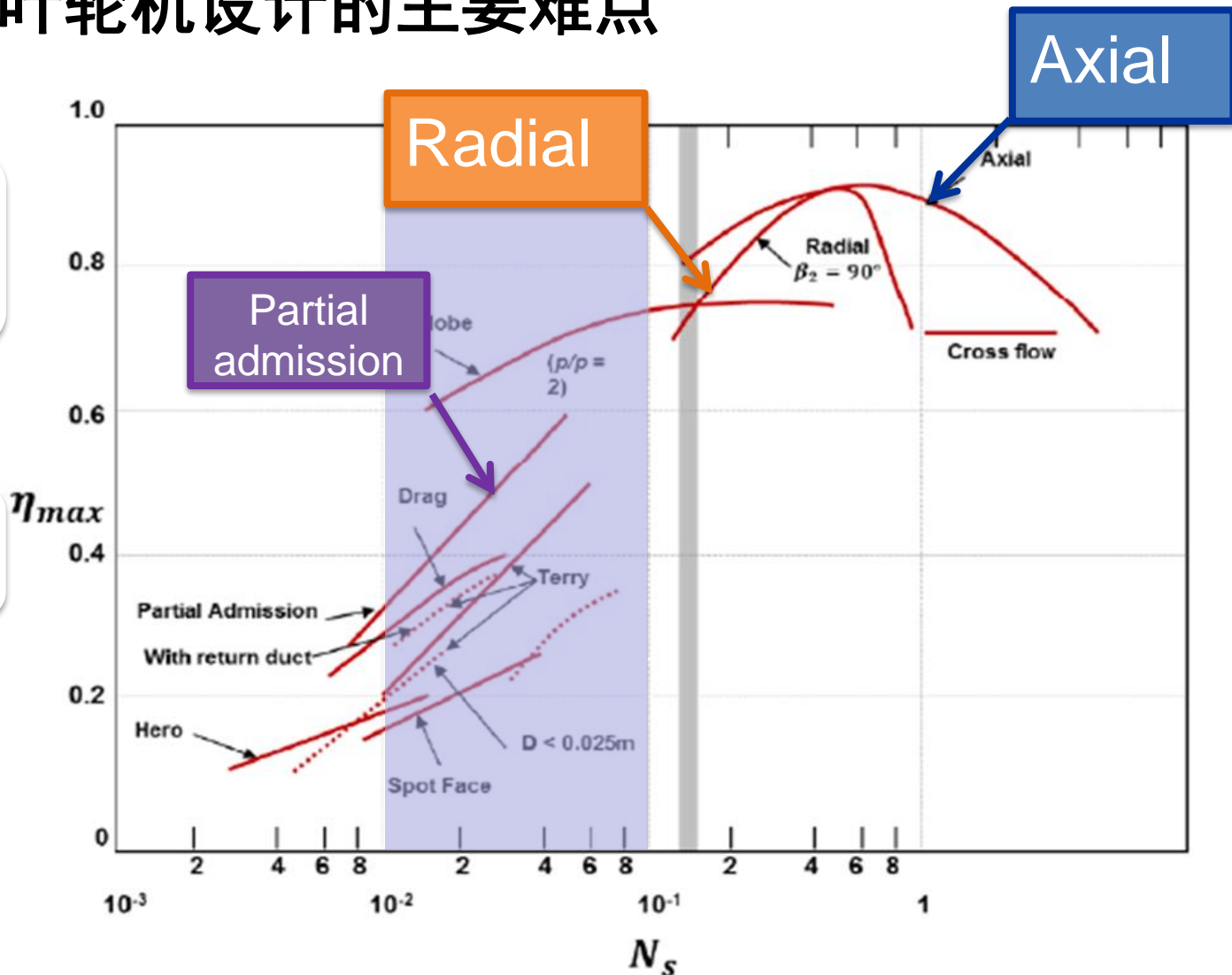
# 超临界CO<sub>2</sub>叶轮机设计的主要难点

小流量  
高焓降



低比转速

$$N_s = \frac{\omega \sqrt{Q}}{\Delta h^{3/4}}$$



# 目 录

---

- 超临界CO<sub>2</sub>动力循环
- 膨胀机涡轮气动设计
- 膨胀机进气蜗壳优化
- 膨胀机涡轮叶型优化



# 膨胀机涡轮气动设计

## ■ 涡轮叶片参数设计

选取流量系数、功率系数与反动度：

$$\phi = \frac{C_m}{U} \quad \varphi = \frac{\Delta h}{U^2} \quad R = 0$$

$$C_{\theta 3} = (1 - R + \varphi / 2)U$$

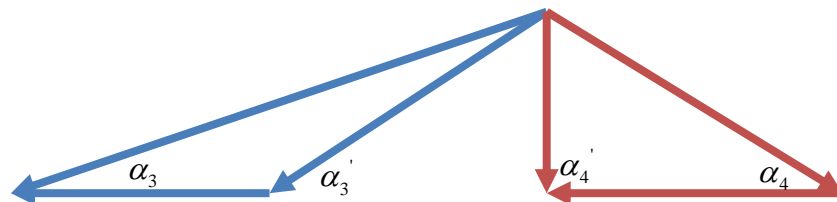
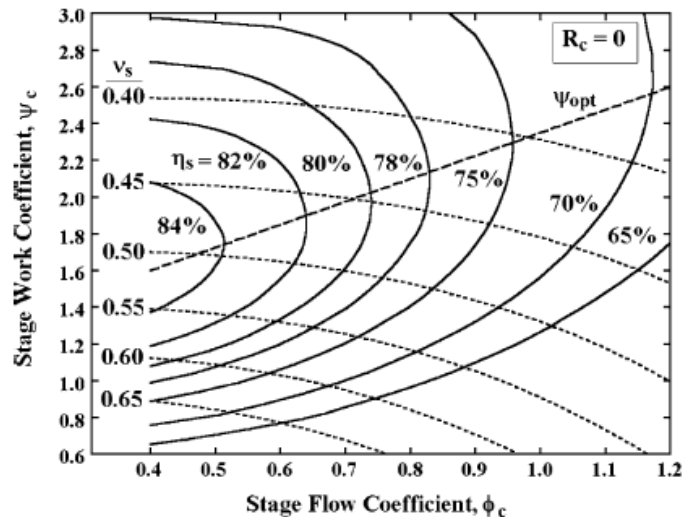
$$C_{\theta 4} = (1 - R - \varphi / 2)U$$

$$\cot \alpha_2 = \cot \alpha_3 = [1 - R + \varphi / 2] / \phi$$

$$\cot \alpha_3' = [-R + \varphi / 2] / \phi$$

$$\cot \alpha_4 = [1 - R - \varphi / 2] / \phi$$

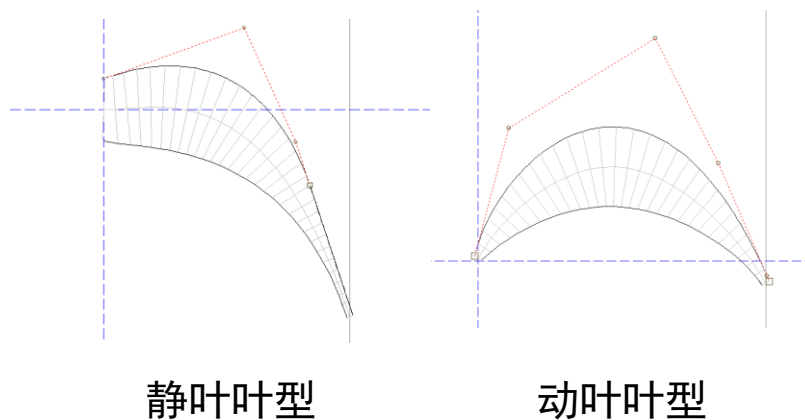
$$\cot \alpha_4' = [-R - \varphi / 2] / \phi$$



速度三角形

## ■ 涡轮叶型初步设计

- 根据速度三角形确定静叶和动叶进出口角度
- 通过贝塞尔曲线控制叶型，实现参数化



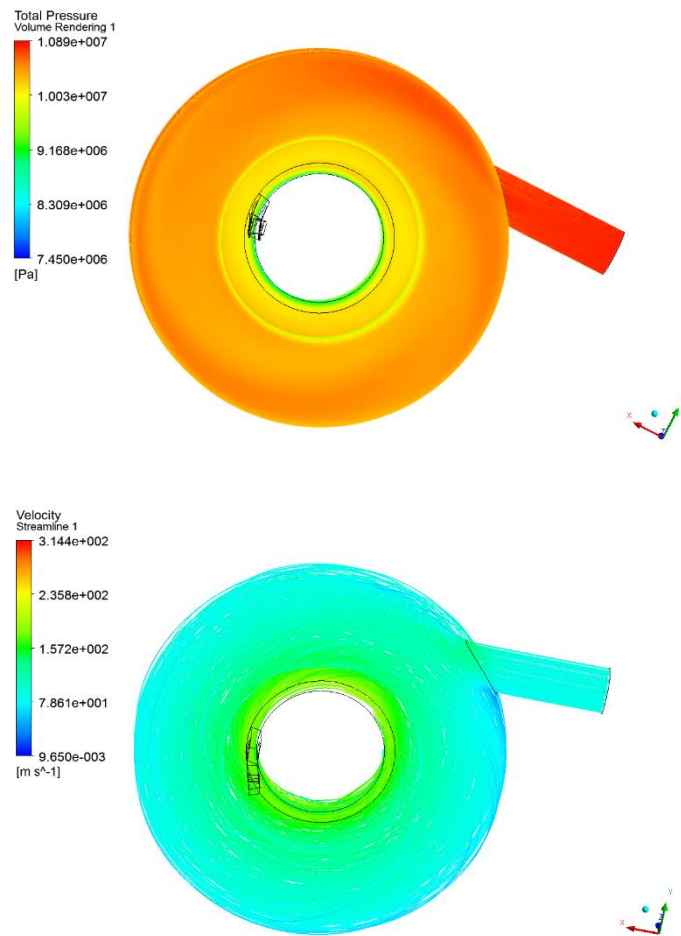
	入口角	安装角	出口角
静 叶	0	40	70
动 叶	60	5	60



## ■ 涡轮性能及流场分析

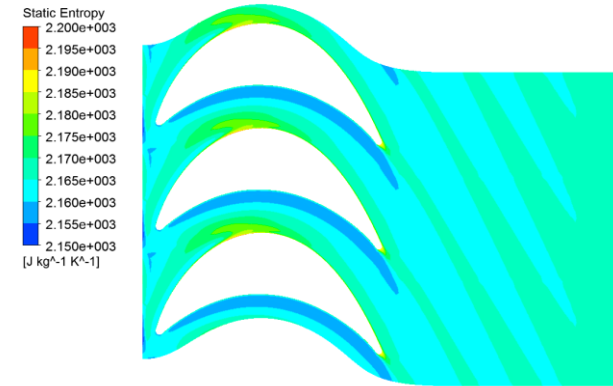
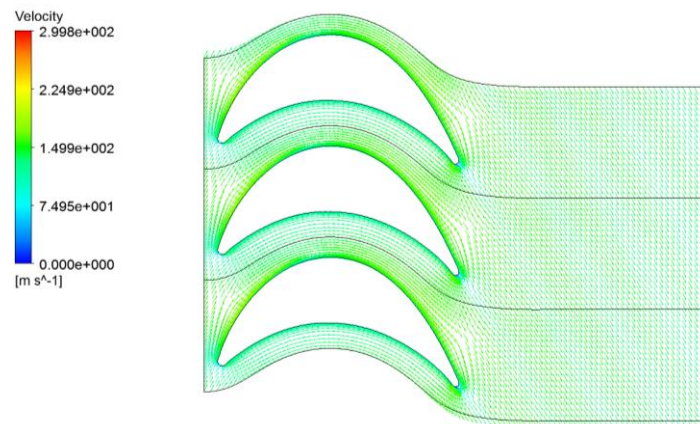
进气度	功率	效率
2.8%	5 kW	56%
16%	30 kW	69%
100%	100 kW	31%

- 全周进气时涡轮效率大幅降低，主要由于进气蜗壳总压损失较大，达到进口总压的 9.8%.

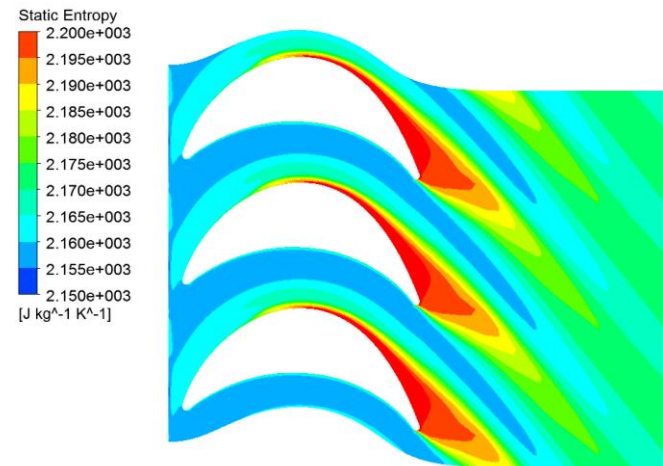
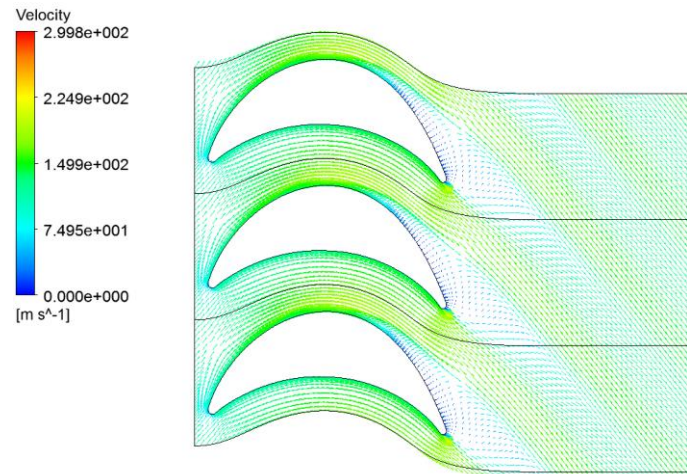


- 动叶叶顶间隙泄漏流动导致吸力面靠近出口附近出现流动分离，流动损失增大

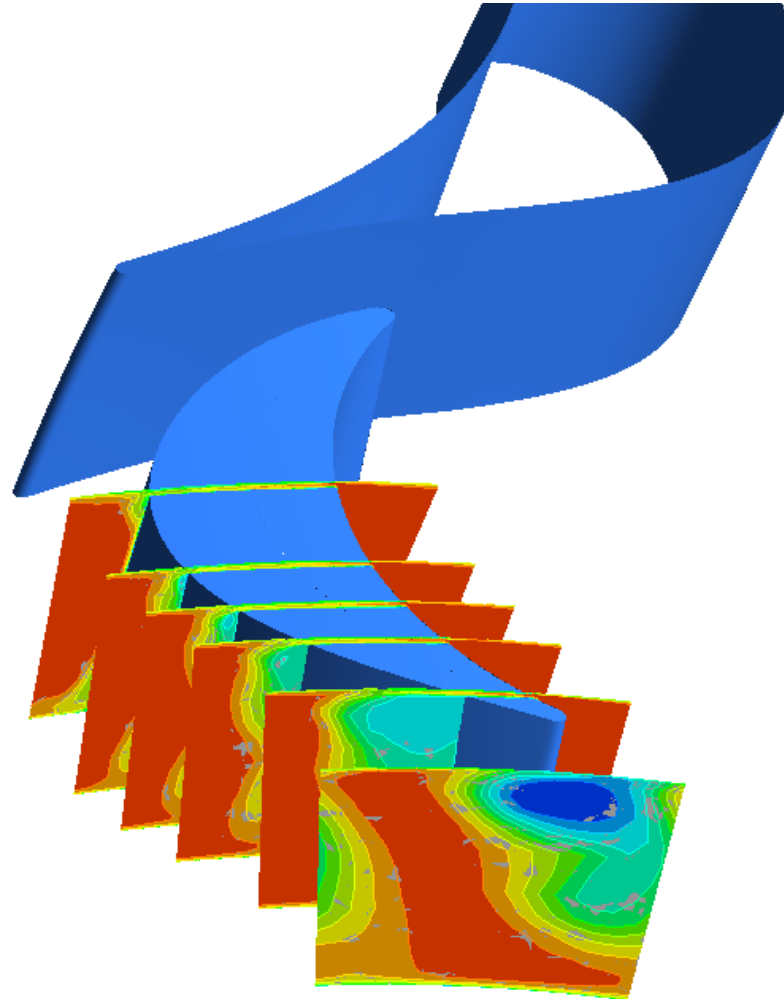
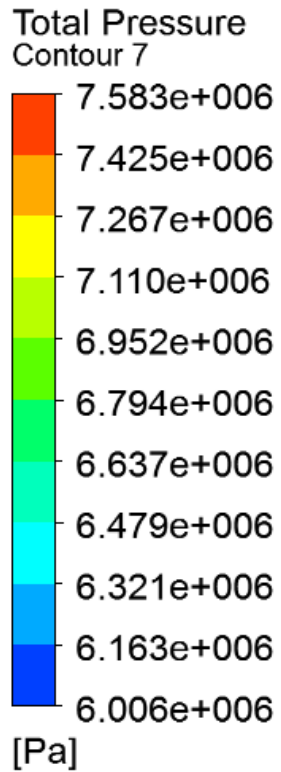
叶根流场



叶尖流场



- 动叶叶顶间隙涡的演变



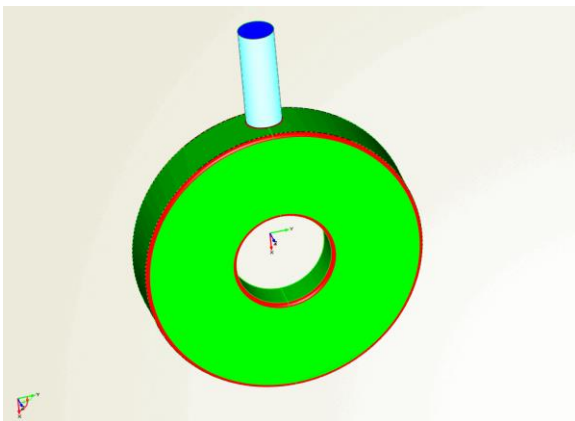
# 目 录

---

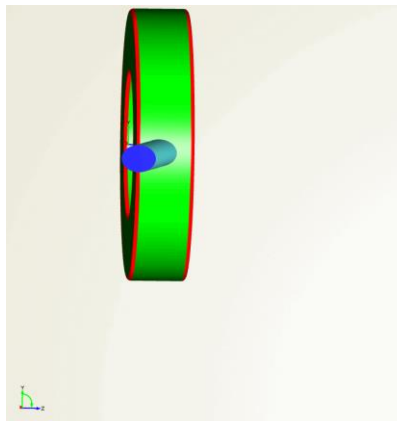
- 超临界CO<sub>2</sub>动力循环
- 膨胀机涡轮气动设计
- 膨胀机进气蜗壳优化
- 膨胀机涡轮叶型优化

# 膨胀机进气蜗壳优化

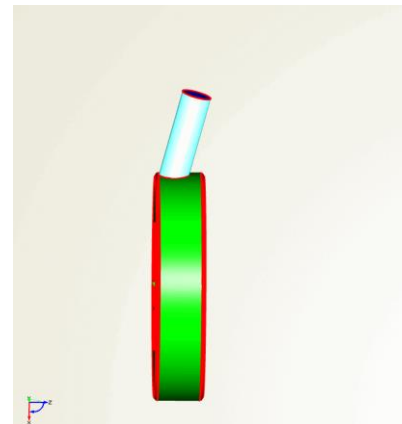
## ■ 进气蜗壳优化参数



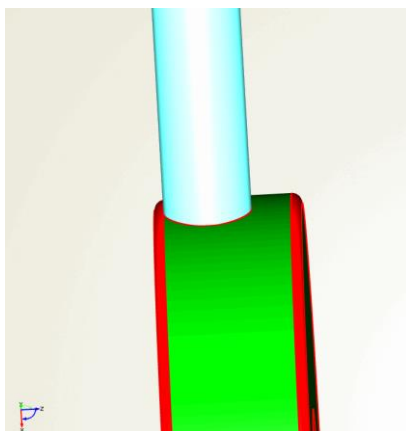
入口切向角度



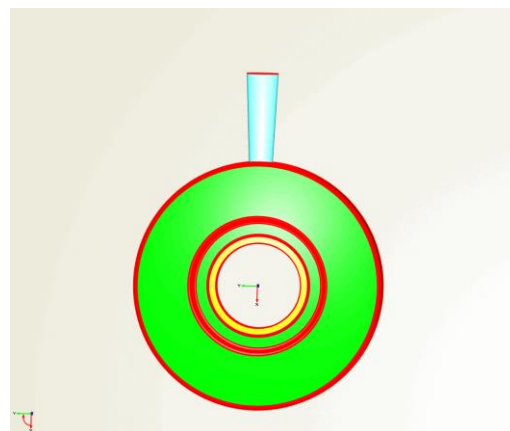
截面形状



入口轴向角度



入口轴向位置



进口管扩张度

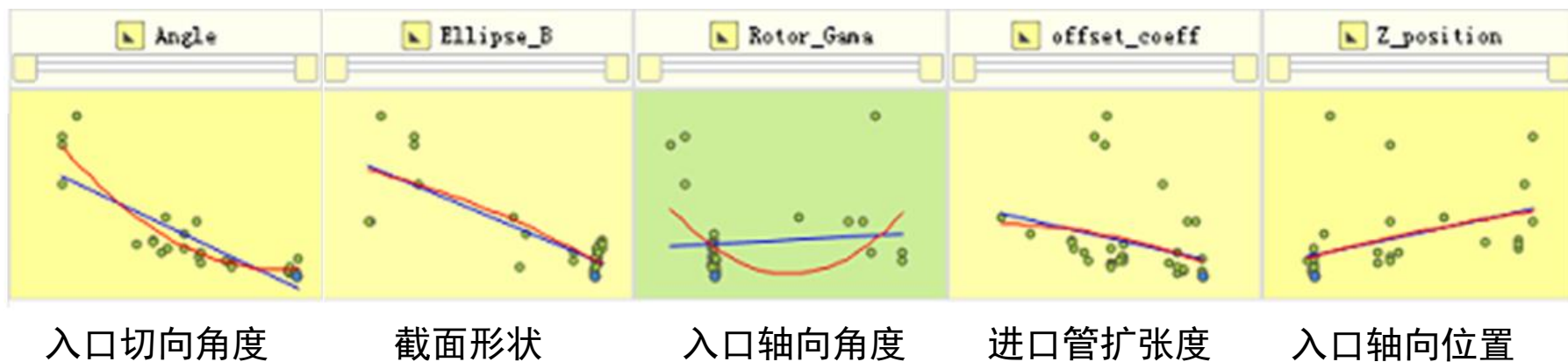
# 优化方法

- 利用CASES软件，采用遗传算法进行优化，经过5代优化，每代8个。采用8核计算平台，耗时26小时。

	Angle	Ellipse_A	Ellipse_B	Rotor_Gama	offset_coeff	Z_position	eval_ResultsValue_00
1	-17.099952	19.25594	19.25594	-5.8062104	1.4750607	5.838011	-0.678
2	-14.944839	14.90779	14.90779	-0.45883879	1.2327031	4.5360243	nan
3	-33.085603	9.9435416	9.9435416	-7.6293584	1.6740627	6.068954	-0.505
4	-8.8370336	19.014755	19.014755	-5.8852522	1.9297139	2.5767773	-0.704
5	-13.489128	13.827909	13.827909	-5.4830243	3.4755621	4.3118869	nan
6	-30.604715	8.3768063	8.3768063	4.4929207	1.7810666	2.8529435	-0.47
7	-15.553216	17.823362	17.823362	6.1029207	3.4050629	3.950325	-0.698
8	-9.3877317	7.8403905	7.8403905	2.7190051	2.5705287	3.8028299	-0.647
9	-14.461433	19.261067	19.261067	-5.8855573	1.810222	5.8397047	-0.692
10	-11.465553	19.009628	19.009628	-5.8059052	1.491426	2.5750835	-0.692
11	-9.3877317	7.7583581	7.7583581	3.6174583	2.4931838	6.0681918	-0.645
12	-33.085603	10.025788	10.025788	-8.5278096	1.7514076	3.8033921	-0.519
13	-33.085603	10.162295	10.162295	-7.6293584	2.2838178	5.9388739	-0.583
14	-14.944839	14.90779	14.90779	-0.45883879	0.84920119	4.6661044	-0.64
15	-8.8507668	19.038682	19.038682	-5.807126	1.9295125	3.7908583	-0.704
16	-17.076219	19.232013	19.232013	-5.8843366	1.4752621	5.3176905	-0.68
17	-20.123598	19.028505	19.028505	-5.8077363	2.4050629	3.9506638	nan
18	-4.2803845	17.835538	17.835538	6.2243076	1.9295125	3.7905196	-0.711
19	-11.641337	15.511284	15.511284	-5.8059052	1.1047013	2.7505562	-0.668
20	-8.6749828	19.038682	19.038682	-5.807126	2.3182371	3.615555	-0.716
21	-3.2119478	19.01561	19.01561	-5.8852522	1.8104738	2.5767773	-0.717
22	-20.098519	19.260212	19.260212	-5.8855573	1.9294621	5.8397047	-0.685
23	6.9835203	18.919905	18.919905	-5.6508736	2.4050629	2.5767773	-0.732
24	-31.37377	17.919066	17.919066	5.9893187	1.9297139	3.950325	nan
25	-3.2119478	19.014755	19.014755	-5.8852522	1.8136965	2.5767773	-0.718
26	-8.8370336	19.01561	19.01561	-5.8852522	1.9282898	2.5767773	-0.705
27	-8.8370336	19.038682	19.038682	-5.807126	1.9297139	2.5767773	-0.704
28	-8.8507668	19.014755	19.014755	-5.8852522	1.9037308	3.7908583	-0.704
29	6.9835203	18.919905	18.919905	-5.6508736	2.4437354	2.5766079	-0.729
30	-4.2803845	17.835325	17.835325	6.2243076	1.89084	3.7908583	-0.712
31	-4.2666514	19.01561	19.01561	-5.7289998	1.6042206	2.4900572	-0.713
32	8.0382238	18.919905	18.919905	-5.807126	2.6112857	2.5767773	-0.736
33	6.9897871	18.919905	18.919905	-5.6508736	3.1398398	2.5767773	nan
34	-8.6612497	19.038682	19.038682	4.1930286	1.5814603	3.615555	-0.7
35	8.0382238	18.919905	18.919905	-5.6508736	2.6112857	2.5747448	-0.731
36	6.9835203	18.919905	18.919905	-5.807126	2.4437354	2.5772854	-0.726
37	8.3897917	18.919905	18.919905	-5.6508736	2.6112857	2.5767773	-0.709
38	6.6319524	18.919905	18.919905	-5.8461891	2.4437354	2.5766079	-0.73
39	-3.2119478	18.974594	18.974594	-5.6508736	2.4050629	2.5767773	-0.721
40	6.9835203	15.24126	15.24126	-5.8852522	1.8136965	2.5767773	-0.723



## ■ 优化结果

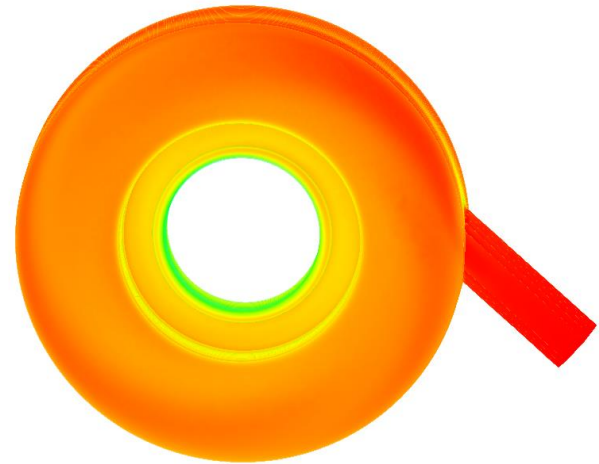


	入口切向角度	截面形状	入口轴向角度	进口管扩张度	入口轴向位置	总压损失
优化前	40	9	0	1	2.45	<b>9.8%</b>
优化后	8.04	18.92	-5.65	2.61	2.57	<b>1.1%</b>

## ■ 优化结果

进气蜗壳形状优化后总压损失显著减小。

Total Pressure  
Volume Rendering 1  
1.080e+007  
9.950e+006  
9.100e+006  
8.250e+006  
7.400e+006  
[Pa]

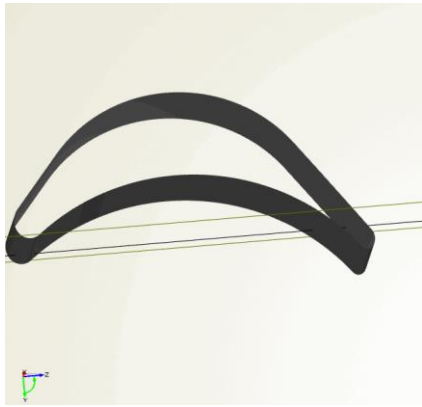


# 目 录

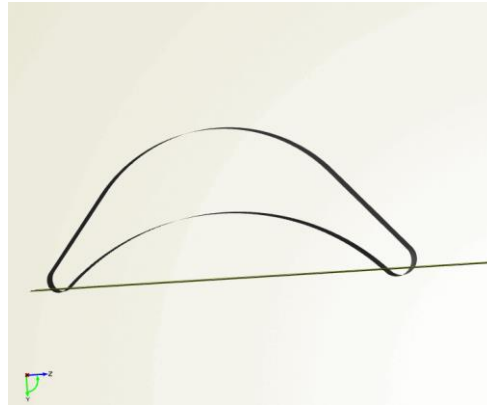
---

- 超临界CO<sub>2</sub>动力循环
- 膨胀机涡轮气动设计
- 膨胀机进气蜗壳优化
- 膨胀机涡轮叶型优化

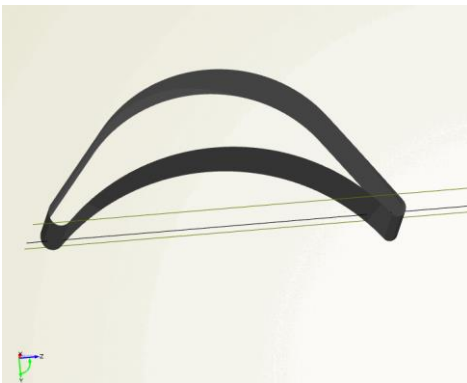
## ■ 叶轮型面优化参数



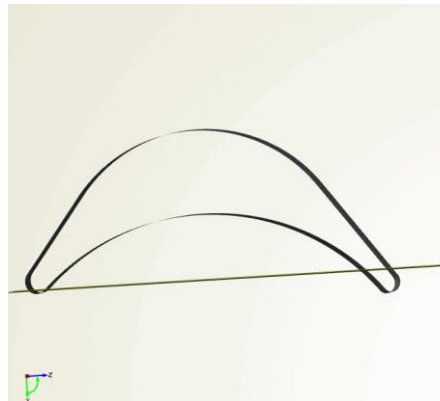
进口型线控制角1



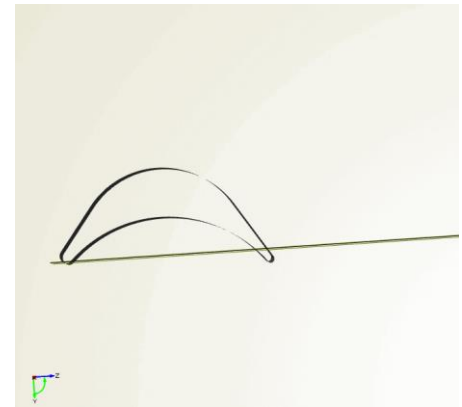
出口型线控制角1



进口型线控制角2



出口型线控制角2



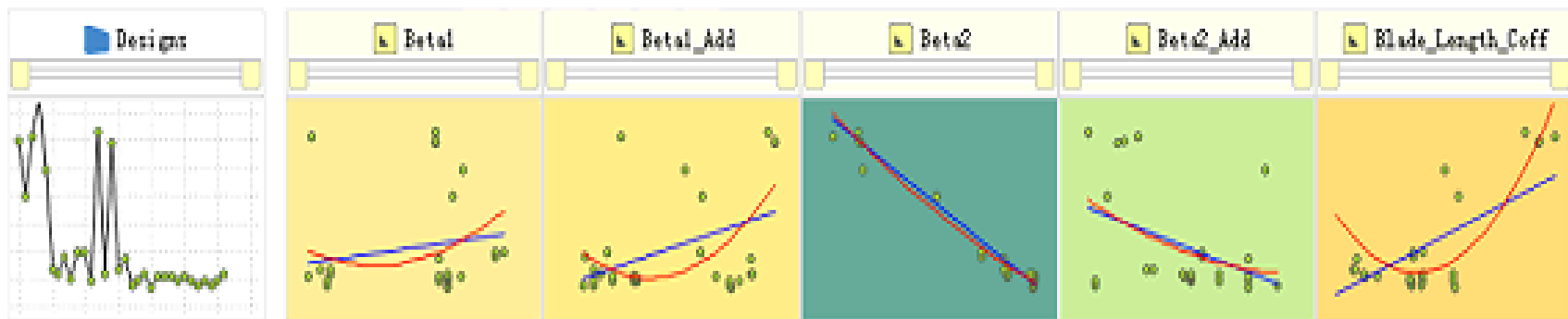
弦长系数

## 优化方法

- 利用CASES软件，采用遗传算法进行优化，经过5代优化，每代8个。采用8核计算平台，耗时20小时。

		Total	Beta1_Add	Beta2	Beta2_Add	Blade_Length_Coff	eval_eff
	Mcpa2_04_des000	43.105909	13.294035	51.234984	0.1335164	0.14165713	-0.747
	Mcpa2_04_des0017	39.060746	9.1084031	51.224002	2.0790417	-0.30315962	-0.746
	Mcpa2_04_des0027	39.060746	0.0007013	51.224002	2.002940	-0.30315962	-0.746
	Mcpa2_04_des0029	43.105909	13.253019	51.235294	9.1335317	-0.42070279	-0.745
	Mcpa2_04_des0011	42.904955	13.295421	51.243229	0.1335164	-0.26022433	-0.743
	Mcpa2_04_des0024	39.072465	9.1084031	51.233760	2.0790417	-0.43629252	-0.743
	Mcpa2_04_des0020	43.105926	10.506157	51.224910	6.1332418	0.12407074	-0.743
	<b>Mcpa2_04_des0030</b>	<b>43.185621</b>	<b>9.7541924</b>	<b>51.224910</b>	<b>0.1332722</b>	<b>-0.42085145</b>	<b>-0.743</b>
	Mcpa2_04_des0018	43.189626	10.410452	51.224910	7.133257	-0.43842985	-0.742
	Mcpa2_04_des0026	42.904955	13.56347	51.243229	0.1296101	-0.26022433	-0.742
	Mcpa2_04_des0008	43.197345	10.451469	51.234654	6.1332418	-0.43842985	-0.741
	Mcpa2_04_des0021	42.916091	10.412161	51.243229	6.1332418	-0.86031126	-0.739
	Mcpa2_04_des0022	38.404486	9.7597946	49.446021	5.8871443	-1.0271687	-0.739
	Mcpa2_04_des0023	43.672145	12.879438	51.229528	0.1287008	-0.28871756	-0.739
	Mcpa2_04_des0025	43.189626	10.451469	51.224910	6.1332418	-0.30329595	-0.739
	Mcpa2_04_des0006	43.189809	13.911681	51.223897	6.1334859	-0.26352026	-0.738
	Mcpa2_04_des0019	43.197528	13.952697	51.233483	6.1334859	-0.26352026	-0.738
	Mcpa2_04_des0013	39.172076	9.3796123	49.428321	5.8790875	-1.0326819	-0.737
	Mcpa2_04_des0005	39.172076	9.1884031	49.428321	4.6201675	-1.1732891	-0.734
	Mcpa2_04_des0015	39.79707	9.1884031	49.193942	4.8790723	-1.1466215	-0.734
	Mcpa2_04_des0016	42.922133	13.911681	49.193837	8.1335164	-1.1272831	-0.727
	Mcpa2_04_des0007	44.829938	8.9201953	47.719005	6.5348151	-0.40327306	-0.724
	Mcpa2_04_des0009	44.830121	12.380407	47.708019	6.5348502	-0.26352026	-0.722
	Mcpa2_04_des0010	45.111192	9.5784553	47.699474	6.5348151	-0.4066901	-0.722
	Mcpa2_04_des0001	43.348043	12.453895	44.541181	3.2914015	0.19137102	-0.681
	Mcpa2_04_des0004	43.736233	11.913654	39.516976	8.7310597	0.0094758526	-0.682
	Mcpa2_04_des0014	42.776013	14.62797	39.428188	3.6206302	1.2206683	-0.643
	Mcpa2_04_des0000	42.776013	14.62797	39.19379	3.8789349	1.2470399	-0.641
	Mcpa2_04_des0002	39.510506	9.9717708	37.370842	4.3613642	1.434287	-0.637
	Mcpa2_04_des0012	42.776013	14.436561	39.119863	2.6206149	1.0360962	-0.634
	Mcpa2_04_des0003	44.976791	14.507378	39.114748	4.9811246	-1.3972076	-0.608
	Mcpa2_04_des0031	43.189809	13.951095	50.922179	7.1335012	0.12407874	pending ...

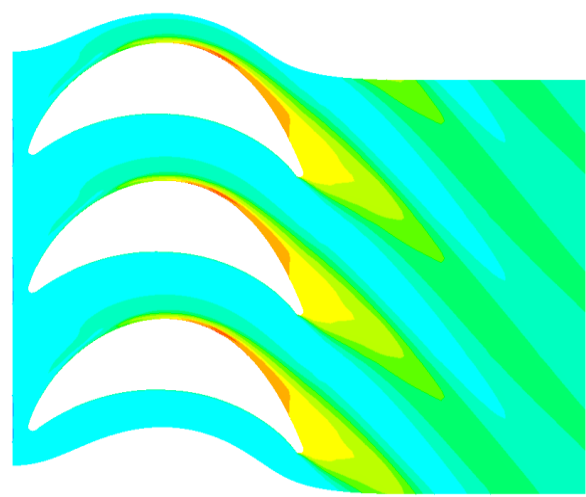
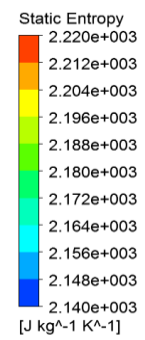
## ■ 优化结果



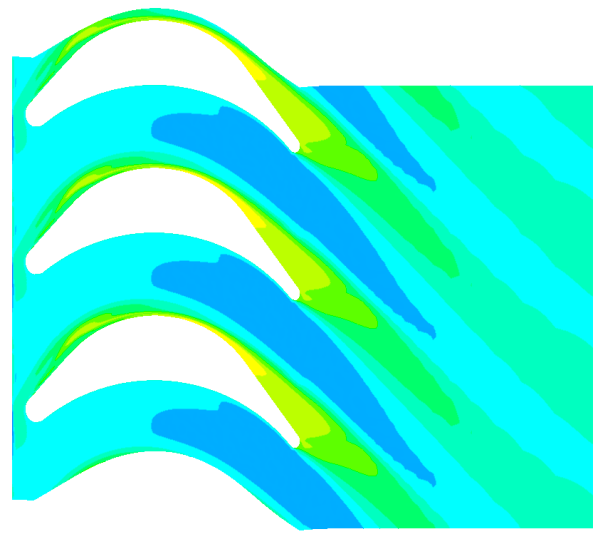
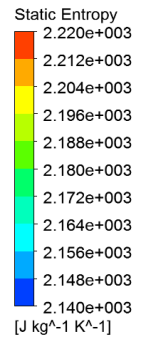
	进口型线 控制角1	进口型线 控制角2	出口型线 控制角1	出口型线 控制角2	弦长系数	效率
优化前	42	20	52	6	-0.55	<b>73.6%</b>
优化后	38	8.1	51.2	2.07	-0.3	<b>74.6%</b>

# 优化结果

优化后叶片吸力面流动损失显著减小。



优化前

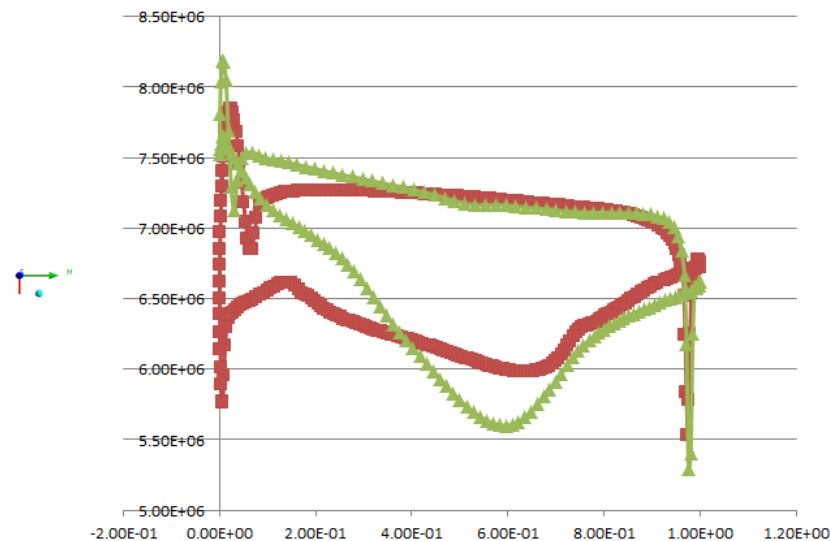
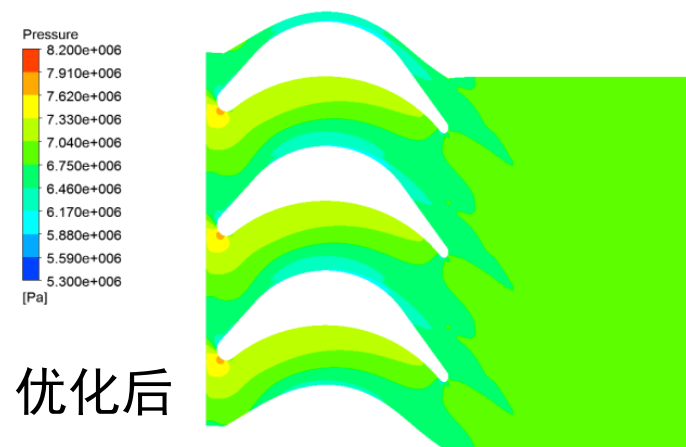
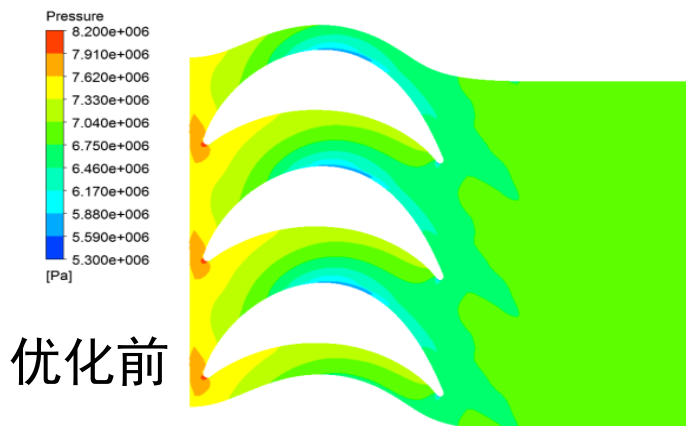


优化后



## 优化结果

优化后叶片压力面和吸力面最大压差减小，泄漏流动降低。

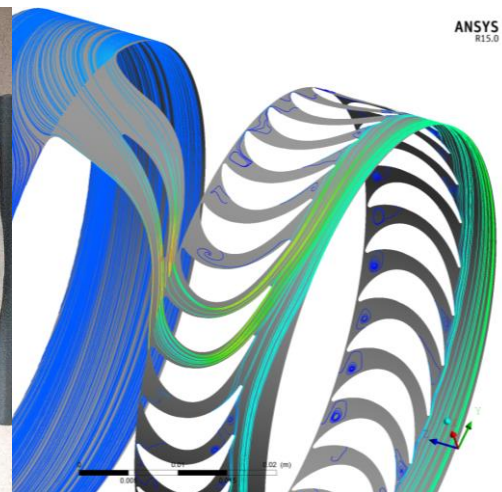
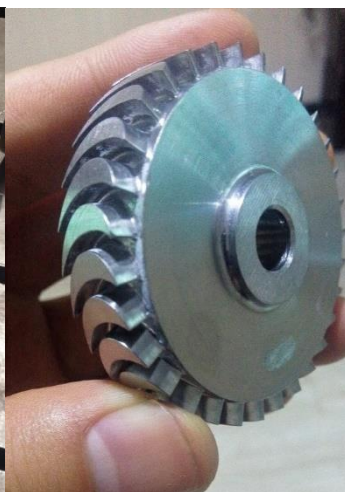
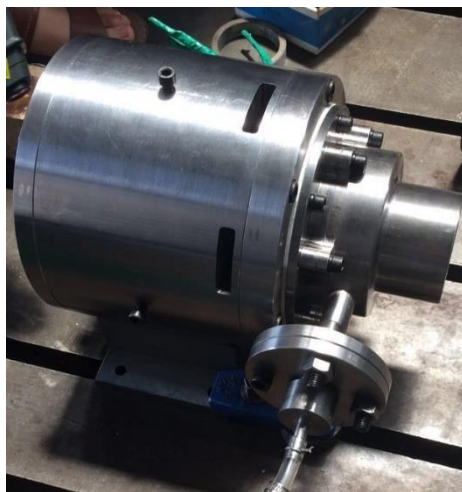
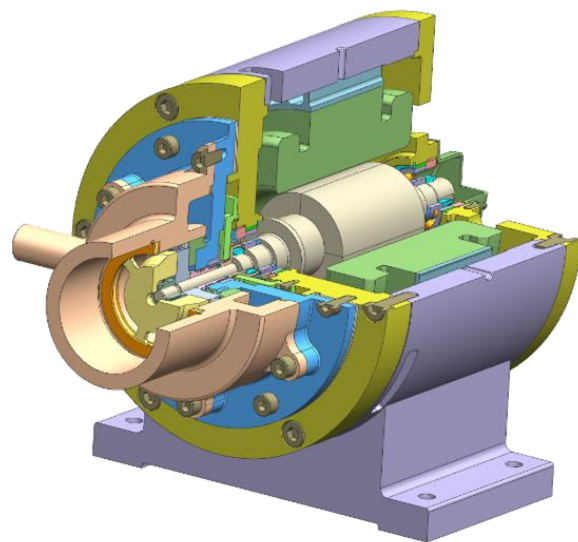




## ■ 超临界CO<sub>2</sub>涡轮膨胀机研制

### 主要参数

- 功率：4.5 kW（可扩展至100kW）
- 转速：40000 r/min
- 膨胀比：1.67
- 流量：0.18 kg/s



---

---

谢 谢!

诸葛伟林

清华大学汽车工程系

zhugewl@tsinghua.edu.cn